Coupled heat transfer to workpiece, wheel, and fluid in grinding, and the occurrence of workpiece burn

ADRIENNE S. LAVINE and TIEN-CHIEN JEN

Mechanical, Aerospace and Nuclear Engineering Department, University of California. Los Angeles, Los Angeles, CA 90024-1597, U.S.A.

(Rewind 23 Mu?, 1989 cud in finu/.fiwm 7 June 1990)

Abstract-A model of heat transfer in grinding was previously developed which predicts the temperature in the grinding zone. This model is used here to predict the occurrence of film boiling of the grinding fluid, and to determine whether or not workpiece burn would subsequently occur. Both film boiling and workpiece burn are assumed to occur at critical grinding zone temperatures. The effects of various parameters are explored, such as fluid and abrasive grain types, and conventional or creep feed grinding conditions.

INTRODUCTION

DURING a grinding process, essentially all of the power supplied to the grinding machine is converted to heat in the region where the wheel contacts the workpiece (the grinding zone) [I]. Grinding fluids are used to lubricate (thereby reducing the grinding power) and to remove heat from the grinding zone. Under some circumstances, the grinding fluid may undergo film boiling, causing a sudden increase in temperature [2-- 61. Depending on a variety of factors, the resulting elevated temperature may or may not cause thermal damage to the workpiece. A model of heat transfer in grinding has previously been developed [7, 81 which predicts the temperature in the grinding zone by considering the coupled heat transfer to the workpiece, wheel, and grinding fluid. In this paper, this model is slightly modified, and is then used to predict when film boiling will occur, and whether it will result in thermal damage to the workpiecc.

Figure I illustrates a grinding wheel in contact with a workpiecc over the grinding zone of length I and depth b (into the page). Some typical grinding parameters are shown in Table I. Note that creep feed grinding differs from conventional grinding in that the workpiece speed, v_w , is much lower, and the depth of cut, a , is much higher (and consequently so is the length of the grinding zone, l). (The remaining quantities in the table will be introduced later.)

A review of the literature reveals a considerable research effort devoted to thermal aspects of grinding. Snoeys et *al.* [9] and Malkin [IO] provide literature reviews. Much of the work [I, I l-141 has concentrated on predicting workpiece surface temperatures in dry grinding, in the absence of significant convective heat transfer. However, convective cooling due to the grinding fluid has been explored by a variety of researchers, see for instance refs. [15–18].

Grinding fluids are especially crucial in creep feed

grinding. Because of the low workpiece speeds and long grinding zones, a point on the workpiece surface remains in the grinding zone longer and therefore tends to be hotter than in conventional grinding. Several papers $[2-5]$ have investigated thermal aspects of creep feed grinding. The results of Shafto et al. [2] suggest that only about 5% of the generated heat remains in the workpiece because a substantial portion of the grinding energy is removed by convection to the fluid. These papers on creep feed grinding have also addressed the phenomenon known as 'surge', in which the power consumption of the grinding machine suddenly fluctuates, associated with metallurgical damage to the workpiece surface. It has been hypothesized that surge is associated with the transition from nucleate to film boiling of the coolant. This hypothesis is supported by the research of Ohishi and Furukawa [5] and Yasui and Tsukuda [6], who showed that when the workpiece surface temperature reaches a value somewhat in excess of 100° C for water based grinding fluid and 300° C for oil, the

FIG. I. The grinding geometry.

NOMENCLATURE

temperature rises rapidly, accompanied by thermal chip is typically not large, and will be neglected here.) damage to the workpiece surface. Thus

ANALYSIS

heat generated at a grain-workpiece interface (q''_{grind}) the fluid, $A_{\text{tot}} - A_{\text{g}}$. conducts into either the workpiece or the abrasive A model has previously been developed of the grain (see Fig. 3). Thus coupled heat transfer to the workpiece, the fluid, and

$$
q''_{\text{grid}} = q''_{\text{wg}} + q''_{\text{g}} \tag{1}
$$

where $q_{\text{wg}}^{\prime\prime}$ is the heat flux into the workpiece at the model will now be summarized (with the workpiece grain location and q''_k the heat flux into the grain. Once model somewhat modified). The separate models for heat enters the workpiece, it may either remain in the heat transfer to the workpiece and fluid build on the workpiece or be removed by convection to the fluid classical theory of moving heat sources (see. for (see Fig. 3). (The rate at which heat leaves with the instance, Jacgcr [191). Consider a rectangular source

$$
q''_{\text{wg}}A_{\text{g}} = q''_{\text{wb}}A_{\text{tot}} + q''_{\text{f}}(A_{\text{tot}} - A_{\text{g}})
$$
 (2)

beginning of

where $A_{\rm g}$ is the total actual grain-workpiece contact Heat is generated in the vicinity of the contacts area, so that the left-hand side is the total rate at which between abrasive grains and the workpiecc (see Fig. heat enters the workpiece. On the right-hand side, 2). In the model, all the heat is assumed to be gen- q_{wb}'' is the heat flux which remains in the workpiece, crated at the grain-workpiece interfaces, although to assumed evenly distributed over the total grinding be more precise, heat is also generated at chip-grain zone area, $A_{\text{tot}} = lb$. Finally, q_i^{ν} is the heat flux into interfaces and at the workpiece-chip shear planes. The the fluid, assumed uniform over the area exposed to

> the abrasive grain [7, 81. One outcome of this model is a prediction of the workpiece temperature. The

Table 1. Typical grinding conditions (in round numbers): $v_s = 30$ m s⁻¹; $d = 200$ mm; $l_e \approx 0.1$ mm $\Rightarrow L_e = v_s l_e / \alpha_s \approx 200$

			v_w (mm s ⁻¹) a (mm) $l \approx \sqrt{(ad)$ (mm) $L = v_s l/\alpha_w$	
Conventional Creep feed	100	0.01		2000 20.000

Heat is generated at grain/workpiece interface, and enters grain (q"_g) and workpiece (q"_{wg}). **Of** heat **which enters workpiece. some remains in** workpiece (q_{"wb}) and some is removed by fluid (q",).

FIG. 3. Heat transfer paths.

of heat (heat flux uniform over the source) moving over the surface of a semi-infinite solid, starting at time $t = 0$. Three simplifications will be made in the models to follow. so as to yield a closed form solution for the temperature in the solid.

(a) A steady-state temperature field has been reached (in a frame of reference fixed to the moving heat source). Based on the formulae presented in Jaeger [19], numerical calculations show that the steadystate assumption is valid provided the heat source has moved a distance of the order of a few source widths, The exact distance which must be traveled depends on the Peclet number, $Pe = vs/\alpha$, where v is the velocity of the source and s the width of the source in the direction of motion. For example, consider a band heat source (infinite in the direction perpendicular to the direction of motion). For $Pe > 8$, the surface temperature at the center of the band will have reached 95% of its steady-state value after the source has moved less than one source width. For $Pe = 0.8$, the source must have traveled 10 source widths.

(b) The rectangular heat source can be approxi-

mated as a band source, infinite in the direction perpendicular to the direction of motion. For $Pe = 8$, the agreement between the surface temperatures for a square and a band heat source appears to be within 12% (based on a figure presented in Jaeger [19]). Agreement improves as the Peclet number increases.

(c) Conduction in the direction of motion can be neglected. This assumption is also best for large Peclet number. For $Pe = 20$, for a band source, the error incurred in neglecting conduction in the direction of motion is less than 13% (except for very near the leading edge of the source, where the temperature is low, anyway).

With these three assumptions, the solution is the well-known error function solution for the temperature distribution in a semi-infinite solid moving in the positive \hat{x} -direction with velocity v , past a uniform heat source at its surface, starting at $\hat{x} = 0$. The solution for the surface temperature rise is

$$
\theta_{s}(\hat{x}) = q'' \sqrt{(4\hat{x}/\pi k \rho c_{p} v)}
$$
\n(3)

where θ_s is the surface temperature rise relative to T_i (the temperature of the solid for $\hat{x} < 0$, before it encounters the source), and q'' the heat flux at the surface. Note that this solution is only valid underneath the source, since beyond the source the temperature rise decays back to zero. A local heat transfer coefficient can then be defined

$$
h(\hat{x}) = q''/\theta_s(\hat{x}) = \sqrt{(\pi k \rho c_p v/4\hat{x})}.
$$
 (4)

Heat transfer to workpiece

Many individual abrasive grains produce heat at discrete points on the workpiece surface. An approximate model of this which has been justified in the past $[11-13]$ is to consider the temperature distribution to be the superposition of a 'background' temperature rise due to a uniform heat source acting over the entire grinding zone, and an 'individual grain' temperature rise which applies only underneath a grain. While the

 $maximum$ workpiece temperature occurs underneath a grain, it is actually the workpiece *background* temperature which determines thermal damage to the workpiece $[13]$. There are two reasons for this: (1) the peak temperature under a grain occurs for a short time, and thermal damage requires time to occur, and (2) the peak temperature occurs in material which will probably be removed, so that it will not affect the quality of the finished surface. However, the peak workpiece temperature under a grain is still of importance, because the grain is continually exposed to this temperature, and therefore this temperature determines tho heat **flus** into the grain.

Background temperature rise. Here, the entire grinding zone is modeled as a uniform source of heat acting over the region $0 < x < l$, with uniform heat flux q''_{wh} into the workpiece. The workpiece moves relative to this heat source with velocity v_w . Since the workpiece is generally many times longer than the grinding zone, the steady-state assumption is valid except for very near the leading edge of the workpiecc. The Peclct number for the workpiece, $v_w l / \alpha_w$, varies greatly between conventional and creep feed prinding. In creep grinding, it is not always large enough to justify using a band source and neglecting conduction in the direction of motion. However, for creep feed grinding. heat transfer to the workpiece is generally a small fraction of the total grinding power, so that a significant error in this portion will not cause a large error in the tcmpcraturc. Then the local heat transfer coctficient corresponding to the workpiccc background temperature i<

$$
h_{\rm wh}(x) = q''_{\rm wh}/\theta_{\rm wh,s} = \sqrt{(\pi (k\rho c_p)_\text{w} r_\text{w}/4x)}.
$$
 (5)

Individual grain temperature rise. This model is a modified version of the one presented in ref. [8]. An individual grain is modeled as a band heat source of width $l_{\rm g}$, causing a heat flux $q''_{\rm wg}$ into the workpiece surface. Relative to an individual grain heat source, the workpiece moves with velocity $v_s - v_w \approx r$, (in the opposite direction from the wheel). Since the grinding zone is usually of the order of a millimeter or longer. and an individual grain-workpiece contact area is of the order of 100 μ m or less, the grain moves at least several times its length. and the steady-state assumption is therefore reasonable. The Peclet number for the grain heat source, $r_s l_g/\alpha_w$, is typically at least 20 or more. so the assumptions of a band heat sourcc and no conduction in the direction **of motion** arc quite good. (These last two assumptions were not made in ref. [8], and the results were within 10% of the approximate solution given here, for $v_s/\sqrt{\alpha_w} > 20.$) The workpiece surface temperature rise due to an individual grain heat **source is** then given by

$$
\theta_{\text{wg},s}(\hat{x}) = q''_{\text{wg}} \sqrt{(4\hat{x}/\pi (k\rho c_p)_{\text{w}} v_s)}.
$$
 (6)

Note that the individual grain temperature rise is given as a function of \hat{x} , the coordinate measured relative to the leading edge of the grain heat **source.** Later. the average temperature underneath a grain heat source will be needed

$$
\bar{\theta}_{\text{wgs}} = (2q_{\text{wgs}}^{\prime\prime}/3)\sqrt{(4l_{\text{g}}/\pi(k\rho c_{\rho})_{\text{w}}v_{\text{s}})}.
$$
 (7)

The heat transfer coefficient corresponding to the average workpiece temperature underneath a grain heat source is

$$
\bar{h}_{\rm wg} = q''_{\rm wh}/\bar{\theta}_{\rm wg,s} = (3/2)\sqrt{(\pi (k\rho c_p)_{\rm w} v_s/4l_g)}.
$$
 (8)

Heat transfer to grinding fluid

The actual contact area between the workpiecc and the grains is typically only a few percent of the total grinding zone area. so that most of the workpiece surface is cxposcd to the grinding fluid. It is assumed that there is a uniform heat flux q_i'' into the fluid from the workpiccc surface. The grinding fluid is assumed to completely fill the space around the wheel grains. to a depth greater than the thermal boundary layer thickness [20]. It is further assumed that the fluid moves past the workpiece with the wheel velocity r_s . Finally, based on the large Peclet number (typically $r_s l / \alpha_r$ is of the order of 10⁵ or greater), a band heat source is used, and conduction in the direction of motion is neglected. The local heat transfer coefficient for the **tluid** is then

$$
h_{\rm f}(x) = q_{\rm f}''/\theta_{\rm f,s} = \sqrt{(\pi (k\rho c_p)_{\rm f} v_{\rm s}/4x)}.
$$
 (9)

This solution is not **valid** if the fluid boils. Later in this paper, it will be assumed that when film boiling occurs, the heat transfer to the fluid becomes ncgligible. i.e. $h_f = 0$.

Heat transfer to abrasive grain

This last model does not build on the classical theory of moving heat sources. A grain moves past the workpiece surface with velocity r_s . If the heat flux into the grain at the workpiece surface is $q_{\rm g}^{\prime\prime}$, and the grain is taken to be a frustum of a cone, then there is an exact solution for the cross-sectionally lumped grain temperature [7]. The local heat transfer **coeffi**cient for the grain is given by

$$
h_{g}(x) = q_{g}''/\overline{\theta}_{g,s} = \sqrt{(\pi (k\rho c_{\rho})_{g}v_{s}/4x)f(\zeta)}
$$
 (10)

where $\zeta(x) = (\pi \alpha_s x / l_g^2 r_s)^{1/2}$, and

$$
f(\zeta) = \frac{2}{\pi^{1/2}} \int_{-\infty}^{\zeta} \frac{\zeta}{\zeta^2} \, \text{erfc}\left(\zeta\right). \tag{11}
$$

Coupling the models

The individual thermal models are coupled by requiring that the surface temperatures match. At a point on the workpiece surface which is cxposcd to the fluid. the workpiece background temperature rise. θ_{wbs} , equals the fluid temperature rise, $\theta_{\text{f,s}}$

$$
\theta_{\text{wbs}}(x) = \theta_{\text{fs}}(x) \Rightarrow \frac{q_{\text{wbs}}^n}{h_{\text{wbs}}(x)} = \frac{q_{\text{f}}^n}{h_{\text{f}}(x)}
$$
. (12a,b)

Underneath a grain, the grain temperature rise, $\theta_{\rm g,s}$. equals the sum of the workpiece background temperature rise and the workpiece temperature rise due to an individual grain

$$
\bar{\theta}_{g,s}(x) = \theta_{\text{wb},s}(x) + \bar{\theta}_{\text{wg},s} \Rightarrow \frac{q''_g}{h_g(x)} = \frac{q''_{\text{wb}}}{h_{\text{wb}}(x)} + \frac{q''_{\text{wg}}}{\bar{h}_{\text{wg}}}.
$$
\n(13a,b)

Taking q''_{grid} in equation (1) to be known, equations (1), (2), (12b), and (13b) are four equations which can be solved for the four heat fluxes ($q_{\rm wb}$, $q_{\rm wg}$, $q_{\rm f}$, and $q_{g}^{\prime\prime}$). Then the four temperature rises $(\theta_{w_{\text{b},s}}, \theta_{w_{\text{g},s}}, \theta_{f,s},\theta_{g_{\text{c},s}})$ and $\bar{\theta}_{\rm g,s}$) are all known as well. An inconsistency now arises. All heat fuxes were taken to be uniform. But then equation (l3b) cannot be satisfied, because the two sides of the equation do not have the same x dependence. This contradiction arises because the actual solution to the coupled heat transfer problem does not have uniform heat fluxes into each of the various components. Some or all of the heat fluxes must depend on x . An approximate method for handling this problem was detailed in ref. [8]. Only the results for the workpiece background temperature will be given here, since as mentioned previously, it is this temperature which governs thermal damage to the workpiece. The workpiece background temperature at the workpiecc surface is given by

$$
\theta_{\text{wb,s}}(x) = \left[1 + \frac{h_{\text{r}}}{h_{\text{wb}}} (1 - A)\right] \left[1 + \frac{h_{\text{g}}}{h_{\text{wg}}}\right] + A(h_{\text{g}}/h_{\text{wb}})
$$
\n(14)

In this expression, A is the fractional grain-workpiece contact area, i.e. $A = A_e/A_{tot}$, and q''_{tot} is the average grinding power flux based on the total grinding zone area, i.e. $q''_{\text{tot}} = q_{\text{tot}}/A_{\text{tot}}$. Here q_{tot} is the total grinding power, i.e. the integral of q''_{grid} over the actual contact area. The two functions \bar{h}_g and (\bar{h}_g/\bar{h}_{wb}) are defined as follows :

$$
\bar{h}_{g} = \frac{1}{l} \int_{0}^{l} h_{g} dx
$$
 (15)

$$
\overline{(h_{\rm g}/h_{\rm wh})} = \frac{1}{l} \int_0^l (h_{\rm g}/h_{\rm wh}) \, \mathrm{d}x. \tag{16}
$$

A non-dimensional temperature rise is now defined

$$
\Phi = h_{\rm wb}(x)\theta_{\rm wb,s}(x)/q_{\rm tot}'
$$
\n
$$
= \frac{1}{\left[1 + \frac{h_{\rm f}}{h_{\rm wb}}(1-A)\right]\left[1 + \frac{h_{\rm g}}{h_{\rm wg}}\right] + A(h_{\rm g}/h_{\rm WD})}.
$$
\n(17)

It has been shown in ref. [8] that Φ is a function of seven parameters

$$
\Phi = \Phi(\kappa_{g}, \kappa_{f}, \alpha_{g}/\alpha_{w}, v_{s}/v_{w}, A, L = v_{s}/\alpha_{w}, L_{g} = v_{s}/\alpha_{w})
$$
\n(18)

where

 $\kappa_{\rm g} = \sqrt{((k\rho c_p)_{\rm g}/(k\rho c_p)_{\rm w})}, \quad \kappa_{\rm f} = \sqrt{((k\rho c_p)_{\rm f}/(k\rho c_p)_{\rm w})}.$ (19)

The first three parameters depend only on the material properties of the workpiece, grain, and fluid. The last four parameters depend as well on operating conditions and wheel geometry. Of all these parameters, *A* and *L,* are least accurately known.

It should be noted that the non-dimensional temperature rise Φ does not depend on x. It can be interpreted in two ways. It is the workpiece surface temperature normalized by the maximum possible surface temperature which would occur if all of the grinding power went into the workpiece, i.e. $\Phi = \theta_{\text{wb}} / \theta_{\text{max}}$ poss, where $\theta_{\text{max.poss.}} = q''_{\text{tot}}/h_{\text{wb}}$. And, it is the fraction of the grinding power which remains in the workpiece, since $\Phi = h_{\rm wb}\theta_{\rm wb,s}/q''_{\rm tot} = q''_{\rm wb}/q''_{\rm tot}.$

From the point of view of predicting film boiling of the coolant and thermal damage, the dimensional temperature will be required. In particular, the maximum value of the workpiece background temperature, at $x = l$, will be used as an indication of whether these phenomena occur.

RESULTS AND DISCUSSION

The results of the analysis were compared to experimental data in ref. [8]. The predictions were excellent for conventional grinding conditions and for creep feed grinding with oil, but were not very good for creep feed grinding with water. Considering the fact that there are no adjustable constants in the model, the agreement is reasonable. The dependence of Φ on the seven non-dimensional parameters was also investigated in ref. [8]. The results showed a strong dependence on the fluid type and abrasive grain type (as quantified by the parameters κ_f , κ_g , and α_g/α_w), and on the velocity ratio v_s/v_w . The remaining parameters $(A, L, \text{ and } L_{s})$ showed smaller, but not negligible, effects. (It should be noted that a decrease in Q, does not necessarily correspond to a decrease in actual dimensional temperature. Recall that $\Phi =$ $h_{\rm wb}(x)\theta_{\rm wb,s}(x)/q''_{\rm tot}$, and both $h_{\rm wb}$ and $q''_{\rm tot}$ depend on the various grinding parameters.)

Film boiling and workpiece burn

Previous experimental studies [5, 6] have shown that when the grinding zone temperature reaches approximately $100-130$ C for water based grinding fluid and 300° C for oil, the temperature of the workpiece suddenly increases. This has been attributed to film boiling of the fluid. When film boiling occurs, it is reasonable to assume that heat transfer to the fluid becomes negligible compared to heat transfer to the wheel and workpiece. Thus, when film boiling occurs, the equations derived previously can be used, with *h,* (or κ_f) set to zero. The following approach can therefore be used to calculate the workpiece temperature. First, it can be calculated assuming the grinding fluid

Table 2. Material properties and non-dimensional parameters

	Water	ΟiΙ	AI. O ₃	CBN	Steel
	0.65	decision is not all company on the product of the 0.15	THE R. P. LEWIS CO., LANSING, MICH. 49-14039-1-120-2	1300	60.5
	1000	820	4000	3450	7854
$c_{\scriptscriptstyle B}$	4180	2000	770	506	434
$K_{\rm F}$	0.115	0.0345			
$\kappa_{\rm g}$			0.829	3.32	
$\alpha_{\rm g}/\alpha_{\rm w}$			0.841		

remains liquid. If the temperature calculated in this way would exceed the temperature at which film boiling has been observed to occur, the temperature can be recalculated assuming there is no fuid present (i.e. dry grinding). (Note that this approach assumes an abrupt transition between no boiling and fihn boiling. and therefore will cause an overestimate of the temperature under conditions for which nucleate boiling would actually occur. This flaw is not extremely important. because if the fluid is undergoing nucleate boiling, the workpiece will remain cool enough to avoid thermal damage, and an accurate knowlcdgc of the tcmpcrature is not crucial.)

If film boiling occurs, the resulting temperature calculated under dry conditions may bc high enough to cause thermal damage to the workpiece material. For instance. 'workpiece burn' is observed to occur at a temperature of approximately $700-800 \text{ C}$ [9, 21]. In the remainder of this paper, the occurrence of workpiece burn will be used as a representative cxamplc of thermal damage.

Figure 4 shows an example of Φ vs $(r_s/v_w)^{1/2}$ for an aluminum oxide wheel. with a water based grinding fluid, oil. and no fluid (dry). These choices determine the values of $\kappa_{\rm g}$, $\kappa_{\rm f}$, and $\alpha_{\rm g}/\alpha_{\rm w}$, assuming a steel workpiece (see Tabie 2). The wear flat area is taken to be

1%, and $L_g = 200$ (see Table 1). Two values of L are considered. corresponding to typical conditions for conventional ($L = 2000$) and creep feed ($L = 20000$) grinding (see Table I). It should be recalled that typical values of the velocity ratio are lower in conventional than in creep feed grinding (see ranges indicated on the abscissa of Fig. 4).

The quantity Φ of course decreases as v_s/v_w increases, since larger v_s/v_w means more heat is removed by the wheel and fluid, relative to the workpiece. The effect of the type of grinding fluid is as expected. A water based grinding fluid is most effective in removing heat from the grinding zone, and therefore yields the lowest value of Φ , followed by oil, with dry grinding yielding the largest value of Φ .

Two numerical cxamplcs will now be considered for typical conventional and creep feed grinding conditions with a water-based grinding fluid. It should be noted that grinding power is actually a dependent variable which is determined by the workpiccc material. wheel and fluid types, and grinding conditions. Generally speaking. grinding power is of the same order of magnitude for conventional and creep feed grinding, but since the grinding zone area is larger by roughly an order of magnitude in creep feed grinding, the grinding heat flux, q''_{tot} , is corres-

FIG. 4. Φ vs $(v_s/v_w)^{1/2}$ for aluminum oxide wheel, with water, oil, and without fluid.

pondingly lower in creep feed grinding. In the following numerical example, the grinding power will be chosen somewhat arbitrarily (although not unrealistically), to force film boiling to occur. The objective is to see whether or not workpiece burn then occurs. Recall that $\Phi = \theta_{\text{wb,s}}/\theta_{\text{max,poss}}$, where $\theta_{\text{max,poss}}$ $q''_{\text{tot}}/h_{\text{wh}}$. For typical conventional grinding conditions (see Table 1), with $q''_{\text{tot}} = 100 \text{ W mm}^{-2}$, it is calculated that $\theta_{\text{max,poss}}(l) = 780^{\circ}\text{C}$. Then from the graph with $(v_s/v_w)^{1/2} = 17$ and $L = 2000$, $\Phi_{\text{water}} = 0.23$, and thus $\theta_{\text{wb,s}}(l) = 180^{\circ}$ C, which is well over the film boiling temperature. Therefore, the temperature should actually be calculated from the upper dashed curve, which gives $\Phi_{\text{dry}} = 0.63$ and $\theta_{\text{wb},s}(l) = 490^{\circ}$ C. At this temperature, the workpiece would not undergo workpiece burn. Next. a typical creep feed grinding case is considered (see Table 1). With $q''_{\text{tot}} = 10 \text{ W mm}^{-2}$, $\theta_{\text{max}, \text{poss.}}(l) = 2500^{\circ}\text{C}$. Then, since $(v_s/v_w)^{1/2} = 170$ and $L = 20000$, $\Phi_{water} = 0.038$ and thus $\theta_{wh,s}(l) = 95^{\circ}\text{C}$, which again is above the film boiling temperature (taking into account an ambient temperature $T_i = 30^{\circ}$ C or so). Thus the temperature should be calculated from the upper solid line $(L = 20000)$, for which $\Phi_{\text{dry}} = 0.28$, $\theta_{\text{wbs}}(l) = 700^{\circ}$ C, and workpiece burn would probably occur.

In the example given above, when film boiling occurred in conventional grinding, workpiece burn did not occur, but when film boiling occurred in creep feed grinding, workpiece burn did occur. These results are fairly typical for grinding with an aluminum oxide wheel [22]. and can be explained as follows. Since workpiece speeds are much lower in creep feed than in conventional grinding, the heat transfer to the workpiece is much lower. Therefore, heat transfer to the fluid is relatively more important, and when it is reduced due to film boiling, the temperature rise is greater in creep feed grinding than in conventional grinding.

These same concepts can be illustrated in a different way. Note that the workpiece surface temperature rise at the end of the grinding zone can be rewritten as follows :

$$
\theta_{\text{wbs},i}(l) = \Phi q_{\text{tot}}''/h_{\text{wb}}(l) = \Phi \frac{2}{\sqrt{\pi}} (v_s/v_w)^{1/2} L^{-1/2} \frac{q_{\text{tot}}''^l}{k_w}.
$$
\n(20)

Thus the grinding power (per unit depth of the grinding zone) is given by

$$
q'_{\text{tot}} = q_{\text{tot}}/b = q''_{\text{tot}}l = \frac{\sqrt{\pi}}{2} k_{\text{w}} (v_{\text{s}}/v_{\text{w}})^{-1/2} L^{1/2} \theta_{\text{wbs}}(l) / \Phi.
$$
\n(21)

Therefore, the critical grinding power corresponding to film boiling of water is

$$
(q'_{\text{tot}})_{\text{fb,water}} = \frac{\sqrt{\pi}}{2} k_{\text{w}} (v_{\text{s}}/v_{\text{w}})^{-1/2} L^{1/2} \theta_{\text{fb,water}} / \Phi_{\text{water}}
$$
(22)

and similarly for oil

$$
(q'_{\text{tot}})_{\text{fb},\text{oil}} = \frac{\sqrt{\pi}}{2} k_{\text{w}} (v_{\text{s}}/v_{\text{w}})^{-1/2} L^{1/2} \theta_{\text{fb},\text{oil}} / \Phi_{\text{oil}}.
$$
 (23)

For workpiece burn to occur. the critical grinding power is

$$
(q'_{\text{tot}})_{\text{burn}} = \frac{\sqrt{\pi}}{2} k_{\text{w}} (v_{\text{s}}/v_{\text{w}})^{-1/2} L^{1/2} \theta_{\text{burn}} / \Phi_{\text{dry}}.
$$
 (24)

These quantities are plotted in Fig. 5 as a function of $(v_s/v_w)^{1/2}$. The graph is for an aluminum oxide wheel. Once again, the values of A and L_e are held fixed, as indicated, and two values of L are presented. The values of $\theta_{\text{fb,water}}$, $\theta_{\text{fb, oil}}$, and θ_{burn} are taken to be 100, 270, and 700°C, respectively. For the parameter values considered here, the critical grinding power for film boiling of oil is usually slightiy greater than that for water. However, this is not always the case, because there are two competing effects at work (see equation (23)). Since the temperature at which oil undergoes film boiling is higher than for water, there is a tendency for the critical grinding power to be higher for oil. On the other hand, the fact that oil does not remove heat from the grinding zone as effectively as water (i.e. Φ is higher) causes the critical grinding power to be lower for oil. As v_s/v_w increases, heat removed by the fluid becomes a larger fraction of the total, and so this latter effect becomes more important. Thus, it can be seen that as v_s/v_w increases, the two curves become closer, and they cross in the case of $L = 2000$. The graph also demonstrates the point made earlier concerning the occurrence of workpiece burn when film boiling occurs. For small v_s/v_w , the critical grinding power for burn is greater than that for film boiling (of either water or oil). Thus when $(q'_{\text{tot}})_{\text{fb}}$ is exceeded, film boiling occurs, but burn does not immediately occur (if it is assumed that the grinding power does not change much when the fluid undergoes film boiling). For large v_s/v_w , the opposite is true. Thus, when $(q'_{\text{tot}})_{\text{fb}}$ is exceeded, film boiling occurs, and immediately causes burn.

Also shown on Fig. 5 are three points taken from experimental results. Yasui and Tsukuda [6] indicated that when the grinding power exceeded about 5.2×10^4 W m⁻¹ for grinding with a water based fluid, with $(v_s/v_w)^{1/2} = 25$, the workpiece temperature suddenly increased to the value it had in dry grinding. but burn did not occur. This point is indicated with an octagon, and is seen to coincide very closely with the curve for film boiling of water (for $L = 2000$, which approximately coincides with the conditions of Yasui and Tsukuda's experiment). Since the curve for workpiece burn lies above the curve for film boiling of water, the workpiece would not be expected to burn under these conditions, as was found in the experiments. Also shown are two points from Ohishi and Furukawa's experiment [5]. They indicated the con-

FIG. 5. Critical grindmg power for tilm boiling and burning, for aluminum oxide wheel.

ditions under which burn occurred for grinding with water based fluid and oil, as shown by the diamond and four pointed star. The point for oil coincides quite closely with the curve for film boiling of oil (for $L = 20000$, which approximately coincides with the conditions of the experiment). Since this curve lies above the curve for workpiece burn, burn would occur as soon as film boiling occurred, in agreement with the experimental results. The point for water does not agree with the curve for film boiling of water. The reasons for this discrepancy arc currently under investigation.

An alternative to aluminum oxide abrasives is to use CBN (cubic boron nitride) or diamond abrasive grains. There arc two advantages of these so-called superabrasives. Because they are very hard, they remain sharp. and thercforc tend to grind with lower grinding power than aluminum oxide abrasives. In addition. both of these materials have a very large thermal conductivity. so that the grains rcmovc more heat from the grinding zone than aluminum oxide grains. It is difficult to determine the thermal conductivity of a single grain. but the values may be as high as 2000 W m $+ K^{-1}$ for diamond and 1300 W m^{-1} K⁻¹ for CBN, as compared to 46 W m⁻¹ K⁻¹ for aluminum oxide [23]. The effect of the thermal properties of CBN will now be illustrated.

Figure 6 shows Φ as a function of $(v_s/v_w)^{1/2}$ for CBN abrasives and water, oil, or dry grinding. This graph demonstrates that Φ is significantly lower for CBN abrasives than for aluminum oxide abrasives (compare to Fig. 4), due to the fact that the CBN abrasives remove a significant amount of heat from the grinding zone. This also results in less of a spread in the values of Φ for water, oil, and dry grinding, since the contribution of the grinding fluid to the heat transfer is comparatively less significant.

Figure 7 shows the critical grinding power values for grinding with CBN abrasives. The CBN abrasives yield higher critical grinding powers for film boiling and burn, due to the lower values of Φ . Over the entire range of v_s/v_w shown, there is no crossover of the curves (for a fixed value of L). That is, over the entire range, when film boiling occurs for either water or oil. workpiece burn would not occur at that same value of grinding power. The lack of crossover in the case of CBN abrasives is because the heat removed by the grinding fluid is relatively less important compared to the heat removed by the CBN abrasives.

CONCLUSIONS

A model of heat transfer in the grinding process has been used to predict the maximum grinding zone temperature. and thereby to explore the occurrcncc of film boiling and thermal damage to the workpiece (i.e. workpiece burn). The occurrence of film boiling was modeled by assuming that when a critical temperature is reached (approximately $100 - 130$ °C for water and 300 C for oil). the heat transfer to the fluid becomes negligible. This of course causes a sharp rise in the workpiecc background tempcraturc. For aluminum oxide wheels, this rise is typically insufticicnt to cause workpiecc burn in conventional grinding, but does cause workpiece burn in creep feed grinding. For CBN wheels, the heat removed by the abrasive grains is significant. As a consequence. higher grinding power is required to cause film boiling, and it is predicted that workpiece burn would usually not occur cvcn after film boiling occurs.

FIG. 6. Φ vs $(v_s/v_w)^{1/2}$ for CBN wheel, with water, oil, and without fluid.

FIG. 7. Critical grinding power for tilm boiling and burning, for CBN wheel.

Acknowledgements-The support of the National Science Foundation, General Motors, General Electric, and Norton Company is gratefully acknowledged. The helpful comments and suggestions of Prof. Stephen Malkin were also gratefully appreciated.

REFERENCES

- I. J. 0. Outwater and M. C. Shaw, Surface temperatures in grinding, *Trans. ASME* 74, 73-86 (1952).
- 2. G. R. Shafto, T. D. Howes and C. Andrew, Thermal aspects of creep feed grinding, 16th Machine Tool

Design Research Conf., Manchester, U.K., pp. 31-37 (1975).

- 3. J. W. Powell and T. D. Howes, A study of the heat flux at which burn occurs in creep feed grinding, 19th MTDR Conf.. Manchester, U.K., pp. 629-636 (1978).
- 4. C. Andrew, Coolant application in creep feed grinding, Int. Conf. on Creep Feed Grinding, Bristol, U.K., pp. 167-183 (1979).
- 5. S. Ohishi and Y. Furukawa. Analysis of workpiece temperature and grinding burn in creep feed grinding, *Bull. JSME* 28(242), 1775-1781 (1985).
- 6. H. Yasui and S. Tsukuda. Influence of fluid type on wet

grinding temperature, Bull. Japan Soc. Prec. Engng 17(2), 133-134 (1983)

- 7. A. S. Lavine, S. Malkin and T. C. Jen. Thermal aspect of grinding with CBN wheels. *Ann. CIRP 38(l). 557 560 (1989).*
- 8. A. S. Lavinc and T. c'. Jen. Thermal aspects of grinding : heat transfer to workpiece, wheel, and fluid. In Collected Papers in Heat Transfer 1989, ASME HTD-Vol. 123, pp. 267-274 (1989). Also, accepted for publication in J . Heat Transfer.
- 9. R. Snoeys, M. Maris and J. Peters. Thermally induce damage in grinding, Ann. *CIRP* 27(2), 571 581 (1978).
- 10. S. Malkin, Grinding of metals: theory and applicatio J. *Appl. Metalworking* 3(2), 95-109 (1984).
- 11. R. S. Hahn, The relation between grinding condition and thermal damage in the workpiece, $Trans. ASME 78$, 807-812 (1956).
- 13. N. R. DesRuisscaux and R. D. Zerkle, Thermal analysis of the grinding process, $J.$ Engng Ind. 92, 428-434 (1970).
- 13. S. Malkin, Thermal aspects of grinding. Part 2—Surface temperatures and workpiece burn, J . Engng Ind. 96, II84 II91 (1974).
- 14. E. M. Kopalinsky, A new approach to calculating the workpiece temperature distributions in grinding. Wear 94,295~322 (19X4).
- 15. N. R. DcsRuisaeaux and R. D. Zerklc. Temperature in

semi-infinite and cylindrical bodies subjected to moving heat sources and surface cooling. *J. Heat Transfer* 92. 456 464 (1970).

- 16. W. J. Sauer, Thermal aspects of surface grinding, $Proc₊$ Int. Grinding Conf., pp. 391-411 (1972).
- 17. D. G. Lee. R. D. Zerkle and N. R. DcsRuisseaux. An cxpcrimental study of thermal aspects of cylindrical plunge grinding, J. Engng Ind. 94, 1206 1214 (1972).
- IX. W. B. Rowe. J. A. Pettit. A. Boyle and J. L. Moruzri, Avoidance ofthermal damage in grinding and prediction of the damage threshold. drw. *ClRP 37(l). 327 330* (1988).
- 19. J. C. Jaeger. Moving sources of heat and the temperature at sliding contacts, *Proc. R. Soc. N.S.W.* **76,** 203-224 (1942).
- **70.** A. S. Lavine, A simple model for convective cooling during the grinding process, *J. Engng Ind.* 110, 1 6 (198X).
- 21. S. Malkin, *Grinding Technology, Theory and Applications* of Machining with Abrasives, Chap. 6. Ellis Horward. Chichcster;Wiley. New York (1989).
- 22. A. S. Lavine and S. Malkin. The role of cooling in creep feed grinding, *Int. J. Adv. Mf. Technol*. 97-111 (1990).
- 23. C. F. Gardinier, Physical properties of superabrasiv *Ceramic_Bull.* **67,** 1006-1009 (1988).

TRANSFERT THERMIQUE COUPLE ENTRE LA PIECE, LE FORET ET LE FLUIDE PENDANT LE PERCAGE ET APPARITION DU BRULAGE DE LA PIECE

Résumé---Un modèle de transfert thermique dans le perçage a été précédemment développé pour prédire la température dans la zone du perçage. Ce modèle est utilisé ici pour prédire l'apparition de l'ébullition en film du fluide et pour déterminer si la pièce brûle ou non. L'ébullition en film et le brûlage de la pièce sont tous deux supposés apparaître dans la zone des températures critiques. Les effets de différents paramètres sont explorés tels que les types de fluide et de grains abrasifs et les conditions conventionnelles ou de grande vitesse de perqage.

GEKOPPELTER WÄRMEÜBERGANG ZWISCHEN WERKSTÜCK, SCHLEIFSCHEIBE UND FLÜSSIGKEIT BEIM SCHLEIFEN UND DAS ENTSTEHEN VON ZUNDER

Zusammenfassung-In einer früheren Arbeit wurde ein Modell für den Wärmeübergang beim Schleifen entwickelt, das die Berechnung der Temperaturverteilung in der Schleifzone erlaubt. Dieses Model1 wird jetzt dazu verwendet, um das Auftreten von Filmsieden im Schleifmittel vorauszuberechnen und um festzustellen, ob infolgedessen am Werkstiick Verzunderungen auftreten werden. Es wird angenommen, da0 sowohl das Filmsieden als such die Verzunderung bei kritischen Temperaturen auftretcn. Der EinfluB verschiedener Größen wird untersucht: Art von Schleifflüssigkeit und -korn, herkömmliche Schleifbedingungen, Schleifen mit geringem Vorschub.

ВЗАИМОСВЯЗАННЫЙ ТЕПЛОПЕРЕНОС К ОБРАБАТЫВАЕМОЙ ДЕТАЛИ, КОЛЕСУ И ЖИДКОСТИ ПРИ ШЛИФОВКЕ И ВОЗМОЖНОСТЬ ПРОГОРАНИЯ ДЕТАЛИ

Аннотация - Разработанная ранее модель теплопереноса при шлифовке позволяет определить температуру в зоне шлифования. В данной работе эта модель используется для предсказания пленочного кипения в шлифовальной жидкости, а также определения возможности последующего прогорания обрабатываемой детали. Предполагается, что как пленочное кипение, так и прогорание детали происходят при критических температурах в зоне шлифования. Исследуются эффекты различных параметров, таких как виды жидкости и абразивных гранул, а также условия шлифовки с обычной или ползучей подачей.